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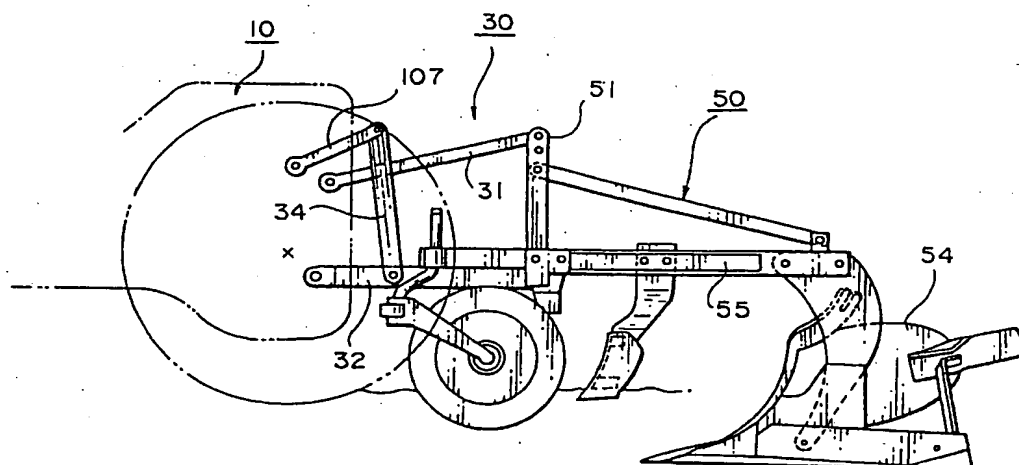
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(54) Cultivating system, tractor and cultivating machine with plow-depth control function

(57) A cultivating system including a tractor (10), a cultivating machine (50) for performing a cultivating work in combination with the tractor, and a link mechanism (30) for linking the tractor and the cultivating machine, including a free-motion mechanism (13) provided to at least one of the tractor, the cultivating machine and the link mechanism, and having a free motion permissible zone in which at least one of the tractor, the cultivating machine and the link mechanism is freely movable relatively to the others in accordance with variation of a cultivation attitude of the cultivating machine due to variation of a cultivation environment, a cultivation-condition control mechanism for detecting the free motion of

at least one of the tractor, the cultivating machine and the link mechanism within the free motion permissible zone in accordance with the variation of the cultivation environment, and moving the cultivating machine in accordance with the free motion permissible zone so as to offset the variation of the attitude of said cultivating machine due to the variation of the cultivation environment, whereby the cultivating machine performs its cultivating work in a desired invariable cultivation condition, and a free-motion transmitting mechanism for transmitting the free motion in the free motion permissible zone to the cultivation-condition control mechanism.

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EP 0 707 783 A1

## Description

### BACKGROUND OF THE INVENTION

#### 1. Field of the Invention

This invention relates to a cultivating system for performing a cultivating work such as plow work using a tractor and a plowing machine while controlling and keeping a plow depth to a desired value irrespective of variation of cultivation environments such as unevenness of the surface of soil, variation of soil type, variation in hardness of soil, etc.

#### 2. Description of Related Art

It has been generally known that a subsoil breaking work is indispensable to make such an improvement of soil that water retentivity is provided not only to a sole pan which locates below a tilth layer, but also to a subsoil layer which locates below the sole pan and has a water-impermeable property through a breaking work of the sole pan and the subsoil layer in a field. For growing of crops, it is ideal that the tilth layer has a water retentivity of keeping a proper amount of water therein and it has a sufficient thickness. When the tilth layer is a little dried and the subsoil contains water roots of crops can extend deeply into the soil and absorb water, so that these crops can avoid suffering from a drought even when rain is lack.

For example in a case where a dry field such as a plowed field, a farm or the like is required to be formed in a wet field such as a paddy field, the dry field must be formed to have a capability of surely excluding water in its tilth layer because crops to be grown in the plowed field are liable to suffer greater damage than paddies when they are immersed in water.

For a paddy field, particularly a paddy field having heavy clay soil, its subsoil is liable to be consolidated due to pressing and puddling of soil by a large tractor, so that soil properties such as gas permeability, water permeability, water retentivity, etc. are degraded. In addition, the tilth layer becomes gradually shallower by a rotary tillage or a cultivating work which is repeated at every year, and thus a water-impermeable sole pan is formed below the tilth layer.

Even when a culvert is provided at the lower side of the sole pan, much rain causes stationary water to occur in the tilth layer, and it keeps the tilth layer in an excessively wet state, that is, in a muddy state. Conversely, When there is no rain for a long time, the sole pan prevents water-absorption of a lower layer portion below the tilth layer, so that the tilth layer is kept in a dry state and suffers drought damage.

Accordingly, when a dry field such as a plowed field is formed in a wet field such as a paddy field, it is required that a sole pair of the dry field, which deteriorates water permeability and water retentivity, is broken, and at the same time a subsoil layer below the sole pan is kept in an expandable and softy state to improve gas permea-

bility and water permeability of soil, and that a cultivation depth is set to a depth suitable for growing of crops in the plowed field, for example, 20 to 30cm.

In view of the foregoing, the following methods have been hitherto adopted. That is, in one method, the sole pan is broken and the subsoil is made expandable and softy by a subsoil breaking machine, and then the tilth layer is plowed and reversed by a plowing machine. In the other method, both of the tilth layer and the sole pan are simultaneously plowed and reversed, and the subsoil layer is made expand and softy.

In a case where a cultivation environment is not homogenous in a cultivating or plowing direction in a field, for example hardness of soil is not constant in the direction of a cultivating work and thus hard and soft areas alternately exists along the cultivation direction in the field, the position of a shear of a plowing bottom is varied in accordance with the variation of hardness of the soil. For example, the shear of the plow is lifted up (floated) at a hard area because the friction resistance between the shear and the soil is large at the hard area. On the other hand, the shear gets deeply into the soil at a soft area because the friction resistance is small at the soft area. Accordingly, an operator on a tractor must control the position of a chisel (plow) through a manual lift operation every time the hardness of the soil varies.

Further, in a case where the soil of the field has an uneven surface in the plowing direction, the cultivating machine such as a plow machine is shifted upwardly or downwardly every time wheels of the tractor passes over a recess or projecting portion, and thus the plowing depth is varied in accordance with unevenness of the soil surface. For example, when the tractor passes over an uneven area, the attitude of the tractor is varied to a backwardly-tilted or forwardly-tilted state. In the former state, the plow machine is forwardly tilted and thus the shear of a bottom plow trends to deeply sting into the soil. In this case, a load imposed on the plow (tractor) is increased. On the other hand, in the latter state, the plow machine is backwardly tilted and thus the shear trends to be floated, and thus the sole pan and the subsoil layer cannot be sufficiently broken.

As described above, in a cultivating work for soil using well-known tractor and cultivating machine such as a plow, subsoiler or the like, the cultivating (plowing) depth is required to be kept to a desired depth at all times. In order to meet this requirement, in addition, to the manual control of the position of the chisel as described above, it has been also proposed that the tractor is provided with a draft function or that a lift mechanism is actuated while measuring the height of the plow from the ground.

Specifically, in one method, the cultivating machine is upwardly and downwardly moved while detecting an upward and downward motion of a gage wheel which moves in contact with the ground, in other words, while detecting an angle between a wheel arm for supporting the gage wheel and a frame of the cultivating machine. In another method, ultrasonic wave or light is emitted to

the ground and a current cultivation or plow depth is measured by detecting ultrasonic wave or light which is reflected from the ground. The cultivating machine is upwardly and downwardly moved on the basis of the measurement result. The former method is representatively disclosed in Japanese Utility Model Application No. 5-1221.

In such a cultivating system a tractor and a cultivating machine such as a plow are linked to each other through a three-point link (hitch) mechanism. The linkage between the tractor and the cultivating machine is regarded as a rigid joint, and thus the joined body of the tractor and the cultivating machine must be treated as one rigid body. Therefore, when an excessive load which exceeds the tractive capacity of a tractor is imposed on the tractor, the following troubles occur in the conventional cultivating system. That is, when such an excessive load is imposed on the tractor, the tractor is kept in such a state that front wheels of the tractor are floated while rear wheels thereof are contacted with the ground. Therefore, the tractor loses its tractive force, and it is impossible for the tractor to continue the cultivating work. On the other hand, when the tractor forcibly continues the cultivating work under such an excessive load in a state where the front wheels thereof are not floated, the cultivating machine (plow machine) may be damaged. In order to prevent such troubles, a tractor is designed into a heavy and rigid body to increase its weight because the tractive capacity of the tractor is proportional to its weight. In addition, with respect to the cultivating machine, it is also designed into a heavy and rigid body, and these heavy and rigid tractor and cultivating machine are linked to each other. As described above, the "excessive-load trouble" as described above can be overcome by increasing the weight of the tractor and the cultivating machine (cultivating system), and thus by increasing the tractive force of the cultivating system.

Those tractors which are used for cultivating works in large-scale fields, for example, in U.S.A., Europe, etc., are generally designed in a relatively large size with a large power. Therefore, the large-scale and large power tractors and cultivating machine suitable for these tractors have been widely used in such countries. However, those tractors which are used for cultivating works in small-scale fields, for example, in Japan, are generally designed in a relatively small size with a small power because such compact tractors are capable of a small sharp turn in relatively small-scale fields. Therefore, the small-scale and small power tractors and cultivating machines suitable for these tractors have been widely used in Japan.

If these tractors and the cultivating machines which are suitably used for small-scale fields are applied to geographically complicated fields, for example, a field whose soil surface is uneven in a cultivating direction, a field having various soil types (variation in hardness of soil) in a cultivating direction, etc., the cultivating work cannot be efficiently performed for the reasons as described above. Therefore, the large-scale and large-

power tractors have been hitherto required to be used in such complicated fields.

## SUMMARY OF THE INVENTION

An object of the present invention is to provide a cultivating system in which an excellent cultivating work can be performed with a large tractive force irrespective of variation of cultivation environments such as unevenness of soil surface, variation of soil types (hardness of soil), etc. even when a relatively light and compact tractor is used, thereby keeping the cultivating depth to a desired fixed value.

Another object of the present invention is to provide a tractor with which the cultivating depth can be kept constant irrespective of variation of cultivating environments to thereby enable an excellent cultivating work.

Further object of the present invention is to provide a cultivating machine to be linked to a tractor, in which the cultivating depth can be kept constant irrespective of variation of cultivating environments to thereby enable an excellent cultivating work.

In order to attain the above objects, according to one aspect of the present invention, a cultivating system using a cultivating machine for performing a cultivating work for soil, a tractor having a hydraulic lift mechanism for upwardly and downwardly moving the cultivating machine and a link mechanism having an upper link and a lower link for linking the tractor and the cultivating machine, includes a free-motion mechanism provided to at least one of the tractor, the cultivating machine and the link mechanism, and having a free motion permissible zone in which the one of the tractor, the cultivating machine and the link mechanism is freely movable relatively to the others in accordance with variation of a cultivation attitude of the cultivating machine due to variation of a cultivation environment, a cultivation-condition control mechanism for detecting a free motion of the one of the tractor, the cultivating machine and the link mechanism within the free motion permissible zone in accordance with the variation of the cultivation environment, and moving the cultivating machine in accordance with the free motion permissible zone so as to offset the variation of the attitude of the cultivating machine due to the variation of the cultivation environment, whereby the cultivating machine performs its cultivating work in a desired invariable cultivation condition, and a free-motion transmitting mechanism for transmitting the free-motion of the one of the tractor, the cultivating machine and the link mechanism in the free motion permissible zone to the cultivation-condition control mechanism.

The cultivation-condition control mechanism may include a cultivation-depth control mechanism for controlling a hydraulic lift mechanism of the tractor to move the cultivating machine upwardly or downwardly in accordance with a shift amount of the one of the tractor, the cultivating machine and the link mechanism on a plane containing a cultivation direction in the free motion permissible zone, so that a cultivation-depth of the culti-

uating machine is controlled to be constant irrespective of the variation of the cultivation environment.

According to another aspect of the present invention, a tractor which is linked through a link unit to a cultivating machine and performs a cultivating work in combination with the cultivating machine, includes a free motion permissible unit which is provided at a mount portion between the tractor and the link unit and through which the link unit is freely movable relatively to the tractor on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of the cultivating machine due to variation of a cultivation environment.

Further, according to another aspect of the present invention, a cultivating machine which includes an erect mast, a body frame and a rear mast for linking the erect mast and the body frame, the cultivating machine being linked through a link unit to a tractor and performing a cultivating work in combination with the tractor, includes a free motion permissible unit which is provided at at least one of the erect mast and the rear mast, and through which the cultivating machine is freely movable relatively to the link unit on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of the cultivating machine due to variation of a cultivation environment.

According to the present invention, when the attitude of the cultivating machine is varied due to the variation of the cultivation environment such as variation of soil type (hardness of soil), unevenness of the surface of soil or the like, this variation is detected by the free-motion unit as a relative movement of at least one of the tractor, the link mechanism and the cultivating machine to the others in the free motion permissible zone of the free-motion unit. This free motion is transmitted through the free motion transmitting mechanism to the cultivation condition (for example, plow depth) control mechanism. The cultivation condition control mechanism controls the draft mechanism having the hydraulic lift mechanism to offset the variation of the cultivation attitude of the cultivating machine, for example, by upwardly or downwardly moving the cultivating machine. Through this operation, the cultivation attitude of the cultivating machine (for example, the plow-depth of the plow machine) can be controlled to be constant.

#### BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a schematic view showing the construction of a cultivation system including a tractor, a cultivating machine and a link mechanism for linking the tractor and the cultivating machine to each other according to this invention;

Fig. 2 shows an embodiment of a free-motion unit according to this invention when the free-motion unit is provided to the tractor;

Fig. 3 shows a modification of the free-motion unit shown in Fig. 2;

Fig. 4 shows another embodiment of the free-motion unit according to this invention when the free-motion unit is provided to an upper link of the link mechanism;

Fig. 5 shows another embodiment of the free-motion unit according to this invention when the free-motion unit is provided to a rear mast of the cultivating machine;

Fig. 6 shows a modification of the free-motion unit shown in Fig. 5;

Fig. 7 shows another modification of the free-motion unit shown in Fig. 5;

Fig. 8 shows an embodiment of a free-motion transmitting mechanism when it is applied to the embodiment shown in Fig. 4;

Fig. 9 shows a modification of the free-motion transmitting mechanism shown in Fig. 8;

Fig. 10 shows another embodiment of the free-motion unit when it is provided to a mast of the cultivating machine;

Fig. 11 is a side view of the free-motion unit shown in Fig. 10;

Fig. 12 shows another embodiment of the free-motion unit when it is provided to the mast of the cultivating machine;

Fig. 13 is a side view of the free-motion unit shown in Fig. 12;

Fig. 14 shows another embodiment of the free-motion unit when it is provided to the mast of the cultivating machine;

Fig. 15 is a schematic diagram showing a cultivation condition control mechanism according to this invention;

Fig. 16 is a schematic diagram of the cultivating machine at a normal state;

Fig. 17A is a schematic view showing the cultivation attitude of the cultivating system when the cultivating machine is forwardly tilted;

Fig. 17B is a schematic diagram showing the shift of the upper link in the free motion zone in the state shown in Fig. 17A;

Fig. 18A is a schematic view showing the cultivation attitude of the cultivating system when the cultivating machine is backwardly tilted;

Fig. 18B is a schematic diagram showing the shift of the upper link in the free motion zone in the state shown in Fig. 18A;

Fig. 19 is a schematic view showing an adjusting motion of the cultivating system to keep the cultivation attitude thereof to a desired fixed state;

Fig. 20 shows a gear system when the free-motion unit comprises a movable mast and it is provided to the mast of the cultivating machine;

Fig. 21 is a cross-sectional view of the gear system shown in Fig. 20;

Fig. 22 is a schematic diagram showing another embodiment of the cultivation condition control mechanism for the embodiment shown in Fig. 20;

Fig. 23 is a schematic diagram showing the cultivating machine at a normal state in the embodiment shown in Fig. 20;

Fig. 24 is a schematic diagram showing the cultivating machine at a forwardly-tilt state in the embodiment shown in Fig. 20; and

Fig. 25 is a schematic diagram showing the cultivating machine at a backwardly-tilt state in the embodiment shown in Fig. 20.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Preferred embodiments according to this invention will be described hereunder with reference to the accompanying drawings.

Fig. 1 shows the whole construction of an embodiment of a cultivating system including a tractor, a cultivating machine such as a plowing machine and a plow-depth controlling device.

In Fig. 1, reference numeral 10 represents a tractor, reference numeral 50 represents a plowing machine (bottom plow) and reference numeral 30 represents a three-point link mechanism. The tractor 10 has a hydraulic circuit with which a lift function for lifting the plowing machine up and down is performed, and it has a lift rod 34 for upwardly or downwardly moving the cultivating machine through a lower link 32 of the three-point link mechanism.

The three-point link mechanism 30 comprises all upper link 31, the lower link 32 and the lift rod 34, and the tractor 10 and the plowing machine 50 are linked to each other through the three-point link mechanism 30. The tractor 10 is also provided with a mount portion to which the three-point link mechanism 30 is secured.

Fig. 2 shows an embodiment of a free-zone unit in which the tractor 10 and the plowing machine 50 are freely movable relatively to each other and with which this relative motion between the tractor 30 and the plowing machine 50, that is, the attitude of the plowing machine 50 is detected and transmitted to a plow-depth control mechanism as described later.

In Fig. 2, a hinge plate 13 is provided at the mount portion of the tractor 10 to which the upper link 31 of the three-point link mechanism 50 is secured. A lower portion of the hinge plate 13 is pivotally supported through a pin 12 by a support member 11 which is provided so as to extend from the end of the tractor backwardly. Two mount holes 14 are vertically formed in the hinge plate 13, and one end of the upper link 31 is selectively fixed to one of the holes 14.

A guide rod 15 is further provided at the upper portion of the hinge plate 13 so as to horizontally extend from the hinge plate 13 and penetrate through an unloaded hole of a guide portion 16 of the tractor 10 so that the guide rod 15 is freely movable along its axial direction. Accordingly, the hinge plate 13 is swingable around the pin 12, and thus the guide rod 15 is freely movable horizontally (in its axial direction) in synchro-

nism with the swing motion of the hinge 13. The end portion of the guide rod 15 is secured to a wire support plate 17 of the tractor 10 through a nut 18. The position of the guide rod 15 on the wire support plate 17 is adjusted by the nut 18. One end portion of a wire 19 for transmitting the motion of the hinge plate 13 is secured to the wire support plate 17, and the other end portion of the wire 19 is passed through a wire guide 19a provided to the guide portion 16 and fixed to an input side of a hydraulic circuit for draft control as described above. In this embodiment, a free-zone area corresponds to a space in which the hinge plate 13, that is, the guide rod 15 is freely movable, and the positional relationship between the tractor 10 and the plowing machine 50, that is, the attitude of the plowing machine 50 is detected and transmitted to a plow-depth control mechanism containing the hydraulic circuit by the hinge plate 13 and the wire 19.

The plow-depth control device of this embodiment includes a free-zone unit comprising the hinge plate 13, the guide rod 15 and the wire support plate 17, a motion transmitting unit comprising the wire 19 and a plow-depth control mechanism which comprises the hydraulic circuit and other elements as described later for performing the draft control.

As described above, the site end of the upper link 31 is linked to one of the holes 14, and the other end thereof is linked to a mast 51 of the plowing machine 50 (cultivating machine). In this embodiment, the relative positional variation between the tractor and the plowing machine is changed to the swing motion of the hinge plate 13, that is, the horizontal positional shift of the guide rod 15. The horizontal positional shift of the guide rod 15 is represented by variation of X, Y and Z shown in Fig. 2, and this variation is transmitted through the wire 19 to the hydraulic circuit. Through the operation of the hydraulic circuit, the attitude of the plowing machine is controlled so that the plow depth is invariable.

Fig. 3 shows another embodiment of the free-zone unit. In this embodiment, in place of the hinge plate 13, a support plate 21 is provided at the mount portion of the tractor 10 for the upper link 31. In this case, two horizontally elongated holes 22 are provided at two stages in the vertical direction, and the upper link 31 is fixed to one of the elongated holes 22 through a pin 23 so that the pin 22 is horizontally movable along the elongated hole 22. A wire 24 is secured to the pin 23 to transmit a motion of the pin 22 to the plow-depth control mechanism.

In the former embodiment, the end portion of the hinge plate 13 to which the upper link 31 is secured is swingable around its supporting point, and this swingable motion of the hinge plate 13 is transmitted to the horizontal motion of the guide rod 15 and then transmitted to the plow-depth control. On the other hand, in the latter embodiment, the hinge plate 13 is not provided, however, since the end portion of the upper link 31 is movable along the elongated hole 22, this horizontal motion of the upper link 31 along the elongated hole 22 is directly transmitted to the plow-depth control mechanism.

Fig. 4 shows another embodiment of the free-zone unit of the plow-depth control device. In this embodiment, the upper link 31 itself is designed to be variable in length along its longitudinal direction. The upper link 31 of this embodiment is divided into two parts, a front upper link 31B and a rear upper link 31A. The end portion of the rear upper link 31A is designed to be insertable into a cylinder portion 31C which is formed at one end portion of the front upper link 31B, and a pin 31D is provided to the side surface of the rear upper link 31A so as to project through an elongated hole 31E formed at the cylinder portion 31C. Therefore, both of the front and rear upper links 31B and 31A are relatively movable over the length of the elongated hole. The pin 31D is fixed to one end of a wire 33 for transmitting the length variation in the longitudinal direction of the upper link 31 to the plow-depth control mechanism containing the hydraulic circuit, and the wire 33 is supported by a guide and extends to the plow-depth control mechanism. That is, the relative shift amount between the front and rear upper links 31B and 31A is transmitted to the plow-depth control mechanism through the wire 33. The end portion of the front upper link 31B is pivotally secured in the hole 14 of the mount portion of the upper link 31 through a pin. Further, the front upper link 31B and the cylinder portion 31C are joined to each other with a screw so as to have a turn-buckle function. Therefore, the whole length of the upper link 31 can be adjusted by the screw-fitting position.

The free-zone units of the above embodiments are provided to the end portion (mount portion) of the tractor for the three-point link mechanism, or the upper link 31 of the three-point link mechanism. However, in the following embodiments, the free-zone unit is provided to the plowing machine itself.

Fig. 5 shows another embodiment of the free-zone unit which is provided to a rear mast 52 of the plow machine 50. In Fig. 5, the plow machine 50 includes a mast 51, a rear mast 52, a frame 55 and a bottom plow 54 which is secured to the frame 55. The rear mast 52 is designed to be variable in length. Specifically, the rear mast 52 comprises an upper rear mast 52A and a lower rear mast 52B, and the lower rear mast 52B is insertable into the upper rear mast 52A. A pin 52C is provided to the side surface of one of the upper and lower rear masts 52A and 52B, and an elongated hole 52F is formed on the side surface of the other rear mast. The upper and lower rear masts 52A and 52B are relatively movably linked to each other so that the pin 52C is projected through the elongated hole 52F and it is slidable along the elongated hole 52F.

The pin 52C is linked to one end portion of a wire 53, and the wire 53 is extended to the hydraulic circuit of the plow-depth control mechanism.

Figs. 6 and 7 show various types of the rear mast 52.

The rear mast 52 shown in Fig. 6 is formed of two flat plates. The rear mast 52 shown in Fig. 6 comprises a flat upper rear mast 52A and a flat lower rear mast 52B each of which is provided with a guide member 52D (52E) having U-shaped section, and these flat upper and

lower rear masts 52A and 52B are linked to each other so that one end portions of the flat rear masts 52A and 52B are insertable into the guide members 52E and 52D of the other flat rear masts. The upper rear mast 52A is provided with an elongated hole 52F, and the lower rear mast 52B is provided with a pin 52C projecting from the side surface thereof. The flat upper and lower rear masts 52A and 52B are linked to each other so that the pin 52C is slidable along the elongated hole 52F over the whole length of the elongated hole 52F while projecting through the elongated hole 52F. Therefore, the rear mast 52A is variable in the longitudinal direction of the elongated hole 52F by a stroke corresponding to the length of the elongated hole 52F.

The rear mast 52 shown in Fig. 7 is formed of two pipe-shaped members. As shown in Fig. 7, the upper and lower rear masts 52A and 52B are designed so that one rear mast is insertable into the other rear mast. In this case, the lower rear mast 52B is designed to be insertable into the upper rear mast 52A. The upper rear mast 52A is provided with the elongated hole 52F, and the lower rear mast 52B is also provided with a through hole 52G. A common pin 52C to which the wire 53 is connected is inserted into the elongated hole 52F and the hole 52G so as to penetrate through these holes 52F and 52G in a state where the upper and lower rear masts 52A and 52B are linked to each other. The motion of the pin 52C is transmitted to the plow-depth control mechanism at the tractor side through the wire 53.

In the above embodiments, the variation of the cultivating conditions is detected and transmitted to the plow-depth control mechanism through the wire. In place of the wire, a differential pressure producing cylinder may be issued to transmit the variation of the cultivating conditions to the plow-depth control mechanism. For example, as shown in Fig. 8, a differential pressure producing cylinder 311 is built in the upper link 31, and a switch valve 103 as described later is directly or indirectly driven by a differential pressure due to fluid produced in the cylinder 311 to upwardly and downwardly the lower link 32.

Alternately, the differential pressure producing cylinder 311 may be provided at the outside of the upper link 31 and in parallel to the upper link 31 as shown in Fig. 10. In this case, the expanding and contracting motion of the upper link 31 is detected as a differential pressure of fluid.

In the above embodiment, the pressure differential cylinder is provided to the free zone unit of the upper link 31 shown in Fig. 4, however, it may be provided to the free zone unit of the tractor shown in Fig 2 and 3 or the rear mast shown in Figs. 5 to 7.

In the embodiments as described above, the free-zone unit is provided to the mount portion of the tractor, the upper link of the three-point link mechanism or the rear mast of the plow machine. In the following embodiment, the free-zone unit is provided to the mast 51 of the plow machine 50.

Fig. 10 is a side view of another embodiment of the free-zone unit of the plow-depth control device, and Fig.

11 is a front view of the free-zone unit. In this embodiment, a movable mast 51A is pivotally supported through a pin 51B at the top end of the mast 51 of the plow machine 50 so as to be movable on a plane containing a plow (cultivating) direction. The motion of the movable mast 51A is restricted by stoppers 58 and 57 which are provided at the front and rear end sides of the mast 51, that is, the movable mast 51A is moveable between the stoppers 58 and 57 in accordance with variation of the cultivating condition. The movable mast 51 is provided with a hole at the top end thereof, and also provided with a wire 53 which is connected to the plow-depth control mechanism. The upper link 31 is secured through this hole to the movable mast 51A, whereby the upper link 31 is linked to the moveable mast 51A, and the motion of the movable mast 51A is transmitted to the plow-depth control mechanism through the wire 53.

Fig. 12 is a side view of another embodiment of the free-zone unit of the plow-depth control device, and Fig. 13 is a front view of the free-zone unit.

In this embodiment, the upper link 31 is not directly linked to the mast 51 of the plow machine, but it is linked to the mast 51 through an intermediate mast 71. The intermediate mast 71 is designed in an arcuate shape as shown in Fig. 13, and a stopper 74 is provided to the top end portion of the intermediate mast 71 so as to project to the plow machine side. The mast 51 is provided with a pin 59, and the stopper 74 is designed to be hooked to the pin 59 from the rear side (plow machine side).

A small gap is kept between a hook 74a of the stopper 74 and the pin 59, so that both of the hook 74a and the stopper 74 are prevented from abutting against each other in a state where the intermediate mast 71 and the mast 51 of the plow machine are erected in parallel to each other. However, when the plow machine 50 is forwardly tilted, the mast 51 abuts against the intermediate mast 71 and pushes it forwardly. On the other hand, when the plow machine 50 is backwardly tilted, the pin 59 of the mast 51 abuts against the hook 74a of the stopper 74, and acts as if it attracts the intermediate mast 71 to the plow machine side.

A seat 75 for securing the lower link 32 to the plow machine 50 through a pin 56 is provided to the lower end portion of the so as to project to the plow machine 50. Accordingly, a space between the hook 74a and the intermediate mast 71 serves as a free zone, and the hook 74a and the intermediate 71 serves as the free-zone unit.

Fig. 14 is a side view of another embodiment of the free-zone unit of the plow-depth control device. In this embodiment, the plow machine 50 has a mast 51 erecting at the central position of the plow-width direction. The mast 51 is provided with an elongated mount hole 51X, and a mount pin 31X is provided to the upper link 31 so as to penetrate through the upper link 31 in a traverse direction. The upper link 31 is linked to the mast 51 of the plow machine 50 by penetrating the mount pin 31X through the upper link 31 and the mount hole 51X to the outside of the mast 51. In this case, the mount pin 31X

(the upper link 31) is horizontally movable along the elongated mount hole 51X in accordance with variation of the attitude of the plow machine 50. The variation of the plow machine 50 is detected by the free zone unit, and transmitted to the plow-depth control mechanism through a wire connected to the pin 31X, whereby the attitude of the plow machine is corrected to a normal state, that is, the plow depth is controlled to be constant.

Next, the plow-depth control mechanism for detecting the variation in the cultivating conditions such as the positional variation at the mount portion of the tractor 10 for the upper link 31 shown in Figs. 2 and 3, the length variation of the upper link 31 shown in Figs. 4, 8 and 9, the variation of the length variation of the rear mast shown in Figs. 5 to 7, the variation of the movable mast 51A shown in Fig. 10 to 13, and the variation of the mount pin 31X joined to the upper link 31 shown in Fig. 14 will be described with reference to Fig. 15.

Fig. 15 is a diagram showing the plow-depth control mechanism having a draft function. The wire 19 (24, 33, 53) through which the variation of the attitude of the plow machine, etc. is transmitted is connected to one end portion of an arm 102 of a rotary sensor 101. The motion of the wire is converted to an electrical signal through the rotary sensor 101. The electrical signal is input to a control box such as a microcomputer to perform a spool switching operation of a switch valve 103 which is provided to a hydraulic circuit for plow-depth control.

The draft function is to finally supply a control signal to the hydraulic circuit for upwardly and downwardly moving the lower link 32. The plow-depth control mechanism includes a hydraulic pipe 106 which extends from a pump 104 through the switch valve 103 to a lift cylinder 105, and transmits a reciprocating motion of the lift cylinder 105 to a lift arm 107, whereby the lower link 32 linked to the end portion of the lift arm 107 is upwardly and downwardly moved. That is, the output signal of the sensor 101 is supplied to the control box such as a microcomputer, and it is used as a trigger signal for switching the switch valve 103 of the hydraulic circuit of the lift cylinder 105 of an elevating mechanism which is installed to the tractor. Further, the output signal is also used as a signal for adjusting the open degree of a flow-amount control valve 44 installed in a hydraulic pipe, and on the basis of this signal, a hydraulic output of the hydraulic pump 104 is supplied to the lift cylinder 105 through the switch valve 103 of the hydraulic circuit formed of hydraulic pipes 106.

The lift cylinder 105 is expanded and contracted by the hydraulic output, and the lift rod 34 is moved by the lift arm 107, so that the lower link 32 is lifted up and pushed down.

In the above embodiment, the sensor 195 is described as a means of outputting an electrical signal, and specifically it may be formed of a load cell, a rotational distortion detector, a potentiometer or the like.

Accordingly, the plow machine which is linked to the lower link 32 is also upwardly and downwardly moved through the free motion in the free motion zone in accordance with the variation of the cultivating conditions.

Next, an actual cultivating (plow) work using the cultivating system as described above when the free zone unit shown in Fig. 14 is representatively used will be described with reference to Figs. 16, 17A, 17B, 18A, 18B and 19.

Fig. 16 is a schematic diagram showing a normal attitude of the plowing machine 50, and Figs. 17A (17B) and 18A (18B) are schematic diagrams of abnormal attitudes (forwardly tilted and backwardly tilted attitudes) of the plowing machine 50: In the figures, Z presents the free zone in which the pin 31X is freely movable along the elongated mount hole 51X in accordance with variation of the cultivating environments. The normal attitude of the plowing machine means that the sectional structure of the soil of a field to be plowed is substantially homogeneous in hardness, and the soil surface is substantially smooth. In this normal state, the tractor is disposed substantially in parallel to the ground, and the plow machine is kept to an originally set attitude, that is, the mast 51 is vertically erected. Accordingly, the plowing work is performed while the pin 31X of the upper link 31 is located at the central and neutral position of the free zone Z, and thus it is disposed in a neutral state without being deviated to the front and rear sides.

When the rear wheels of the tractor pass over an recess portion as shown in Fig. 17A (or the front wheels pass over a projecting portion) or the plow machine abuts against a soft soil area (i.e., the surface soil is soft), the plow machine tends to be forwardly tilted as shown in Fig. 17B, in other words, the tip portion of the plow (shear point) tends to deeply sting into the soil, and thus the pin 31X in the free zone is shifted to the right end of the mount hole 51X (the rear side of the plow direction) in the free zone. This shift of the pin 31X is transmitted to the plow-control mechanism as described above through the wire, and on the basis of the shift amount, the plow-depth control mechanism as shown in Fig. 15 lifts up the lower link 32 so that the tip portion of the plow 54 is kept to a horizontal attitude and the plow depth is made constant.

On the other hand, when the front wheels of the tractor pass over a recess portion as shown in Fig. 18A (or the rear wheels pass a projecting portion) or the plow machine abuts against a hard soil area, the plow machine tends to be backwardly tilted as shown in Fig. 18B, in other words, the tip portion of the plow (shear point) tends to be floated from the soil, and thus the pin 31X in the free zone is shifted to the left end of the mount hole 51X (the front side of the plow direction) in the free zone. This shift of the pin 31X is transmitted to the plow-depth control mechanism as described above through the wire, and on the basis of the shift amount, the plow-depth control mechanism presses down the lower link 32 so that the tip portion of the plow 54 is kept to a horizontal attitude as shown in Fig. 19 and the plow depth is made constant.

Through the plow-depth control operation as described above, the cultivating system of this invention is kept so that the plow depth is invariable at all times

irrespective of unevenness of the soil surface or variation of soil types (hardness, etc.).

Of course, the same plow-depth control operation is performed for not only the embodiment shown in Fig. 14, but also the other embodiments as described above.

In the above embodiments, the variation of the plowing machine is detected and transmitted to the rotary sensor of the plow-depth control mechanism through the wire or the differential pressure producing unit. In the following embodiment, the variation of the plowing machine is detected and transmitted through a gear system to the rotary sensor.

Fig. 20 is a side view of the free-zone unit of this embodiment, and Fig. 21 is a front view of the free-zone unit which is viewed from the side I-I. Thus embodiment has the same construction as shown in Fig. 14 except that a movable mast and a gear system are provided to the mast 51. The same elements as the embodiment shown in Fig. 14 are represented by the same reference numerals.

In Fig. 20, as described with reference to Fig. 14, the mast 51 is provided with the elongated mount hole 51X through which the upper link 31 joined to the tractor 10 is linked to the plow machine 50, and also provided with the mount pin 31X so that the mount pin 31X penetrates through the mount hole 51X and the upper link 31. The mount hole 51X is designed to be elongated in a plow direction, so that the mount pin 31X is freely movable along the elongated mount hole 51X on a plane containing the plow direction. In this case, this freely-movable space of the mount pin 31X also serves as a free zone.

The mast 51 is also provided with a movable arm 180 on the outside surface thereof. The movable arm 180 has a mount hole 181 at one end portion thereof, and it is pivotally supported through a support shaft 190 to the outside surface of the mast 51 so that the pin 31X penetrates through both of the mount holes 51X and 181. Accordingly, the movable arm 180 is freely swingable around the support shaft 190 by an amount corresponding to the whole length of the elongated mount hole 51X in synchronism with the horizontal motion of the pin 31X along the mount hole 51X.

The lower portion of the mast 51 of the plowing machine 50 is provided with securing portions 14 to be secured to the lower link 32 at both sides thereof in the plow with direction. The tractor suspends the lower link 32 through the lift rod 34, and the lift rod 34 is upwardly and downwardly moved by rotation of the lift arm 107 as described later.

The plow-depth control device of this embodiment is substantially equal to those of the embodiment shown in Fig. 14 except that information on the motion of the pin 31X is transmitted to the plow-depth control mechanism through a gear system as described later.

As shown in Figs. 20 and 21, a gear G1 is fixed to the support shaft 190 through a spacer S1 coaxially with the support shaft 190. One side face of the gear G1 which is opposite to the other side face confronting the support shaft 190 is secured through a spacer S2 to a



bracket 192 which is disposed in parallel to the mast 51. The bracket 192 is disposed at the outside of and in parallel to the mast 51 and fixed to the mast 51 through bolts 193 at all upper end position and an intermediate position thereof.

Another gear G2 which is engaged with the gear G1 and has a smaller number of teeth than the gear G1 is provided to the bracket 192 through a support shaft 191. The support shaft 192 has one end fixed to the gear G2 and the other end fixed to the bracket 192, and penetrates through a spacer 194. The support shaft 192 serves to keep constant the distance between the gear G2 and the bracket 192. Further, a rotary sensor 195 is secured to the end portion of the support shaft 191 to convert the rotational (swinging) motion of the movable arm 180 around the support shaft 190 to an electrical signal.

The rotary sensor 195 has a flange 195a at the outer peripheral thereof, and two elongated adjustment holes 195a are formed on the periphery of the flange 195b to adjust the securing position of the sensor 195 to the bracket 192. That is, the flange 195b of the sensor 195 is secured to the bracket 192 by screwing the bolts 196 into the bracket 192 at any position within the elongated adjustment holes 195a. Therefore, the attitude of the flange 195b (sensor 195) can be adjusted in a range corresponding to the length of the elongated adjustment holes 195a. With this adjustment, an output line for the electrical signal of the sensor 195 can be selected to any position.

Fig. 22 is a schematic diagram showing the plow-depth control mechanism of this embodiment, and this plow-depth control mechanism is identical to that of Fig. 15 except for the construction of the rotary sensor. Like the plow-depth control mechanism shown in Fig. 15, in this embodiment, the output signal of the sensor 195 is supplied to the control box such as a microcomputer, and it is used as the trigger signal for switching the switch valve 103 of the hydraulic circuit of the lift cylinder 105 of the elevating mechanism which is installed to the tractor. Further, the output signal is also used as the signal for adjusting the open degree of the flow-amount control valve 44 installed in the hydraulic pipe, and on the basis of this signal, a hydraulic output of the hydraulic pump 104 is supplied to the lift cylinder 105 through the switch valve 103 of the hydraulic circuit formed of hydraulic pipes 106.

The lift cylinder 105 is expanded and contracted by the hydraulic output, and the lift rod 34 is moved by the lift arm 107, so that the lower link 32 is lifted up and pushed down.

In the above embodiment, the sensor 195 is described as a means of outputting an electrical signal, and specifically it may be formed of a load cell, a rotational distortion detector, a potentiometer or the like.

Next, an actual cultivating work when the free zone unit shown in Figs. 20 and 21 is used will be described with reference to Figs. 23 to 25.

As described above, Fig. 23 shows the normal attitude of the plowing machine, and Figs. 24 and 25 show the abnormal attitudes (forwardly tilted and backwardly tilted attitudes) of the plowing machine 50. The normal attitude of the plowing machine means that the sectional structure of the soil of a field to be plowed is substantially homogeneous in hardness, and the soil surface is substantially smooth. In this normal state, the tractor is disposed substantially in parallel to the ground, and the plow machine is kept to an originally set attitude, that is, the mast 51 is vertically erected. Accordingly the plowing work is performed while the pin 31X of the upper link 31 is located at the central position of the free zone Z, and thus it is disposed in a neutral state without being deviated to the front and rear sides.

When the rear wheels of the tractor passes over a recess portion or the plow machine abuts against a soft soil area (i.e. the surface soil is soft), the plow machine tends to be forwardly tilted as shown in Fig. 24, in other words, the tip portion of the plow (shear point) tends to deeply sting into the soil, and thus the pin 31X in the free zone is shifted to the right end of the mount hole 51X (the rear side of the plow direction) in the free zone.

At this time, the movable arm 180 is clockwise rotated in Fig. 20, and an electrical signal corresponding to the rotational amount of the movable arm 180 is obtained by the sensor 195. That is, the rotation of the movable arm 180 is transmitted through the gear G1 to the gear G2, and at this time the rotation of the movable arm 180 is transmitted to the sensor 195 while amplified by a gear group of the gears G1 and G2, so that even a slight variation in attitude of the movable arm 180 can be accurately transmitted to the sensor 195. Therefore, the attitude control can be performed with high precision.

The signal output from the sensor 195 is supplied to the control box to be processed, and then supplied to the control valve 144A of the hydraulic circuit and the switch valve 103, whereby the abnormal attitude of the plow machine shown in Fig. 24 is restored to the normal attitude shown in Fig. 23. That is, the lift cylinder 105 is contracted to clockwise rotate the lift arm 107, so that the lift rod 34 is lifted up. This operation is continued until the attitude of the plowing machine 50 is returned to the attitude shown in Fig. 23, that is, the pin 31X is returned to the central and neutral position in the free zone. The correction of the attitude of the plowing machine 50 is detected on the basis of the output of the sensor 195.

On the other hand, when the front wheels of the tractor passes over a recess portion or the plow machine abuts against a hard soil area, the plow machine tends to be backwardly tilted as shown in Fig. 25, in other words, the tip portion of the plow (shear point) tends to be floated from the soil, and thus the pin 31X of the upper link is shifted to the left end of the mount hole 51X (the front side of the plow direction) in the free zone. In this case, the movable arm 180 is counterclockwise rotated, and this counterclockwise swinging motion is transmitted through the gears G1 and G2 to the sensor 195.

The sensor 195 outputs a signal representing the counterclock motion of the movable arm 180.

On the basis of the above signal, the switch valve 103 is switched, and the open degree of the control valve 144A which corresponds to this signal is selected to control the hydraulic circuit. For example, the lift cylinder 105 is expanded and the lift arm 107 is clockwise rotated around the support point to push the lift rod 34 down, so that the lower link 32 is pushed down and the bottom plow is corrected to a floating state.

This attitude correcting work is continued until the mount pin 31X is returned to the central and neutral position in the free zone Z, whereby the bottom plow is kept at a constant height within a predetermined range at all times. In other words, irrespective of variation of cultivating conditions such as variation in hardness of soil, unevenness of the ground, etc. in a field, the cultivating work and be performed while the bottom plow (cultivating machine) 16 can be set to a predetermined range of plow depth.

This shift of the pin 31X is transmitted to the plow-control mechanism as described above through the wire, and on the basis of the shift amount, the control mechanism process down the lower link 32 so that the tip portion of the plow 54 is kept to a horizontal attitude as shown in Fig. 23 and the plow depth is made constant.

Through the plow-depth control operation as described above, the cultivating system of this invention is kept so that the plow depth is kept invariable at all times irrespective of unevenness of the soil surface or variation of soil types (hardness etc.).

According to this embodiment, even a slight variation of the attitude of the plowing machine is detected as a rotation of the movable arm (shift of the mount pin), and the rotation is amplified by the amplifying means (gear system) and then supplied to the sensor. Therefore, the status of the cultivating environment of the field and the attitude of the plowing machine can be surely detected to control the attitude of the plowing machine with high precision. In addition, even a slight rotation of the movable arm is amplified, so that the control can be finely performed to keep the plow depth in a predetermined range at all times with high precision.

As is apparent from the foregoing, according to this invention, the relative positional (attitude) variation between the tractor and the plowing machine is transmitted to the lift mechanism of the tractor as a signal detected in the free zone, and on the basis of the detected signal, the plowing machine is controlled to be upwardly or downwardly moved, so that the plow depth can be kept constant. In addition, the control of the attitude of the plowing machine makes the tractive force larger or smaller, so that the tractive force suitable for the cultivating work can be obtained.

In the above embodiments, the cultivating system of this invention is applied to control the attitude of the cultivating machine in the vertical direction, that is, in the depth direction. However, this invention is not limited to the control of the cultivating machine in the depth direc-

tion, but may be applied to the control of the attitude of the cultivating machine in the other directions (horizontal direction, etc.), or to the control of the attitude of the other elements other than the cultivating machine.

## Claims

1. A cultivating system using a cultivating machine for performing a cultivating work for soil, a tractor having a hydraulic lift mechanism for upwardly and downwardly moving the cultivating machine and a link mechanism having an upper link and a lower link for linking the tractor and the cultivating machine, including:

a free-motion mechanism provided to at least one of said tractor, said cultivating machine and said link mechanism, and having a free motion permissible zone in which the one of said tractor, said cultivating machine and said link mechanism is freely movable relatively to the others in accordance with variation of a cultivation attitude of said cultivating machine due to variation of a cultivation environment;

a cultivation-condition control mechanism for detecting a free motion of the one of said tractor, said cultivating machine and said link mechanism within the free motion permissible zone in accordance with the variation of the cultivation environment, and moving said cultivating machine in accordance with the free motion permissible zone so as to offset the variation of the attitude of said cultivating machine due to the variation of the cultivation environment, whereby the cultivating machine performs its cultivating work in a desired invariable cultivation condition; and

a free-motion transmitting mechanism for transmitting the free-motion of the one of said tractor, said cultivating machine and said link mechanism in the free motion permissible zone to said cultivation-condition control mechanism.

2. The cultivating system as claimed in claim 1, wherein said cultivation-condition control mechanism includes a cultivation-depth control mechanism for controlling said hydraulic lift mechanism to move said cultivating machine upwardly or downwardly in accordance with a shift amount of the one of said tractor, said cultivating machine and said link mechanism on a plane containing a cultivation direction in the free motion permissible zone, so that a cultivation-depth of said cultivating machine is controlled to be constant irrespective of the variation of the cultivation environment.
3. The cultivating system as claimed in claim 2, wherein said free-motion mechanism comprises a hinge plate which is provided at a mount portion of said tractor for the upper link and is linked to the upper link of said link mechanism, and a guide rod

having one end fixed to said hinge plate and the other end which is secured to said tractor so as to be freely slidable in the cultivation direction, so that said tractor and said cultivating machine is relatively movable through the slidable motion of said hinge plate in accordance with the variation of the attitude of said cultivating machine.

4. The cultivating system as claimed in claim 2, wherein said free-motion mechanism comprises a support plate at a mount portion of said tractor for the upper link, said support plate having a horizontally elongated hole serving as the free-motion permissible zone, and said upper link is secured through the elongated hole to said support plate with a pin so that said support plate and said upper link are freely movable relatively to each other in accordance with the variation of the attitude of said cultivating machine.
5. The cultivating system as claimed in claim 2, wherein said upper link comprises a first link which is secured to said tractor at one end portion thereof, a cylinder having an elongated hole serving as the free motion permissible zone on the side surface thereof and secured to the other end of said first link, and a second link which is secured to said cultivating machine at one end portion thereof and has a projecting pin at the other end portion thereof, said second link being freely slidably inserted into said cylinder so that the pin is projected from the elongated hole so as to be slidable along the elongated hole in accordance with the variation of the cultivation attitude of said cultivating machine.
6. The cultivating system as claimed in claim 2, wherein said cultivating machine comprises a front erect mast, a body frame and a rear mast which links said front erect mast to said body frame and comprises a first rear mast and a second rear mast which are engagedly insertable into each other, one of said first and second rear masts having a pin projecting from the side surface thereof, and the other having an elongated hole serving as the free motion permissible zone through which the pin is projected so as to be freely slidable therealong, whereby said first and second rear masts are linked so as to be relatively movable in the free motion permissible zone in accordance with the variation of the cultivation attitude of said cultivating machine.
7. The cultivating system as claimed in claim 6, wherein said first and second rear masts are designed in a tube shape.
8. The cultivating system as claimed in claim 6, wherein said rear mast comprises first and second flat plates each having a guide member having U-shaped section, and linked to each other so that one

end portion of each of said first and second flat plates is mutually insertable into the guide member of the other, and one of said first and second flat plates has a pin projecting from the side surface thereof while the other has an elongated hole serving as the free motion permissible zone through which the pin is projected so as to be freely slidable therealong, whereby said first and second flat plates are linked so as to be relatively movable in the free motion permissible zone in accordance with the variation of the cultivation attitude of said cultivating machine.

9. The cultivating system as claimed in claim 2, wherein said cultivating machine comprises an erect mast, a body frame and a movable mast which is pivotally supported at one end portion thereof to said mast so as to be movable on a plane containing the cultivation direction and is linked to said upper link at the other end portion thereof, and stoppers which are provided at the front and rear end sides of said mast and serve to restrict free motion of said movable mast, whereby said movable mast is movable between said stoppers in accordance with variation of the cultivation attitude of said cultivating machine.
10. The cultivating machine as claimed in claim 2, wherein said cultivating machine comprises an erect mast, a body frame and an intermediate mast through which said upper link and said mast are linked, said intermediate mast being provided with a stopper having a hook at the tip portion thereof so as to project toward said cultivating machine, and said mast having a pin to which said stopper is hooked so that a gap serving as the free motion permissible zone is kept between said hook and said pin, whereby said mast is movable between said intermediate mast and said hook of said stopper in accordance with the variation of the cultivation attitude of said cultivating machine.
11. The cultivating machine as claimed in claim 2, wherein said cultivating machine includes an erect mast which is linked to said upper link, and a body frame linked to said erect mast, said erect mast being provided with an elongated mount hole serving as the free motion permissible zone, and a mount pin which is penetrated through said upper link in a traverse direction and said elongated mount hole to link said upper link to said mast so that both of said mount pin and said upper link are movable along said elongated mount hole in accordance with the variation of the cultivation attitude of said cultivating machine.
12. The cultivating system as claimed in claim 1, wherein said free-motion transmitting mechanism comprises a wire through which said free-motion mechanism and said cultivation-condition control

mechanism are mechanically connected to each other to transmit the free-motion in the free motion permissible zone of the said free-motion mechanism as a shift motion thereof, and a rotary sensor which is connected to said wire and serves to convert the shift motion of said wire to an electrical signal representing the free motion in the free motion permissible zone of said free-motion mechanism and supply the electrical signal to said cultivation-condition control mechanism.

13. The cultivating system as claimed in claim 1, wherein said free-motion transmitting mechanism comprises a differential pressure producing cylinder which is connected to said free-motion mechanism, and serves to convert the free motion of said free-motion mechanism to a differential pressure and transmit the differential pressure to said cultivation-motion control mechanism.
14. The cultivating system as claimed in claim 13, wherein said differential pressure producing cylinder is built in or secured to the outside of said free-motion mechanism.
15. The cultivating system as claimed in claim 1, wherein said free-motion transmitting mechanism comprises a gear mechanism which is directly engaged with said free-motion mechanism so that the free motion in the free motion permissible zone of said free-motion mechanism is directly converted to a rotational motion of said gear mechanism, and a rotary sensor which is connected to said gear mechanism and serves to convert the rotational motion of said gear mechanism to an electrical signal representing the free motion in the free motion permissible zone of said free-motion mechanism and supply the electrical signal to said cultivation-condition control mechanism.
16. The cultivating system as claimed in claim 15, wherein said cultivating machine includes all erect mast, a body frame and a movable mast which is pivotally supported at one end portion thereof to said mast through a support pin and linked to said upper link at the other end portion, thereof so that the other end portion of said movable mast is movable on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of the cultivating machine, and said gear systems includes a first gear which is fixed to said support pin and a second gear which has a smaller number of teeth than said first gear and is fixed to said rotary sensor, said two gears being engaged with each other.
17. The cultivating system as claimed in claim 1, wherein said cultivation-condition control mechanism includes a control unit for receiving information on the free motion in the free motion permissible

zone of said free-motion zone and outputting a control signal, and a switch valve for controlling oil supply from a pump to said hydraulic lift mechanism on the basis of the control signal from said control unit so that said cultivating machine is returned to all original cultivation attitude through the upwardly and downwardly moving operation of said cultivating machine by said hydraulic lift mechanism.

18. The cultivating system as claimed in claim 1, wherein the variation of the cultivation environment corresponds to unevenness of the surface of soil, variation of soil type, or variation in hardness of soil.
19. The cultivating system as claimed in claim 1, wherein the variation of the cultivation attitude of said cultivating machine corresponds to the tilt of said cultivating machine on a plane containing a cultivation direction relatively to a horizontal direction.
20. A tractor which is linked through a link unit to a cultivating machine and performs a cultivating work in combination with the cultivating machine, including a free motion permissible unit which is provided at a mount portion between said tractor and said link unit and through which said link unit is freely movable relatively to said tractor on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of said cultivating machine due to variation of a cultivation environment.
21. The tractor as claimed in claim 20, wherein said free motion permissible unit comprises an expandable and contractible unit within a predetermined length in the cultivation direction in accordance with the variation of the cultivation attitude of said cultivating machine.
22. The tractor as claimed in claim 21, said expandable and contractible unit is formed integrally with said link unit.
23. A cultivating machine including an erect mast, a body frame and a rear mast for linking said erect mast and said body frame, said cultivating machine being linked through a link unit to a tractor and performing a cultivating work in combination with the tractor, including a free motion permissible unit which is provided at at least one of said erect mast and said rear mast, and through which said cultivating machine is freely movable relatively to said link unit on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of said cultivating machine due to variation of a cultivation environment.
24. The cultivating machine as claimed in claim 23, wherein said free motion permissible unit is provided to said rear mast, and comprises an expandable and

contractible unit wherein a predetermined length in the axial direction thereof in accordance with the variation of the cultivation attitude of said cultivating machine.

tion of the cultivation attitude of said cultivating machine corresponds to variation of a plowing depth.

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25. The cultivating machine as claimed in claim 24, wherein said expandable and contractible unit is formed integrally with said rear mast.
- 10
26. The cultivating machine as claimed in claim 23, wherein said free motion permissible unit is provided to said erect mast, and comprises a movable mast which is linked to said link unit at one end thereof, and is pivotally mounted to said erect mast at the other end thereof so that the one end thereof is freely movable in accordance with the variation of the cultivation attitude of said cultivating machine. 15
- 20
27. A cultivation condition detecting device for use in a cultivation system including a tractor having an upper link, and a cultivation machine which has a mast linked through the upper link to the tractor and performs a cultivation work in combination with the tractor, comprising:
- 25
- a movable mast which is pivotally supported at one end portion thereof to said mast through a support pin and linked to said upper link at the other end portion thereof so that the other end portion of said movable mast is movable together with said upper link on a plane containing a cultivation direction in accordance with variation of a cultivation attitude of the cultivating machine; 30
- a gear mechanism which is connected to said support pin and rotates together with said support pin, said gear mechanism converting the free motion of said upper link to a rotational motion; and 35
- a rotary sensor which is connected to said gear mechanism and serves to convert the rotational motion of said gear mechanism to an electrical signal representing the free motion of said upper link in accordance with the variation of the cultivation attitude of said cultivating machine, so that the variation of the cultivation attitude of said cultivating machine is controlled to be suppressed on the basis of the electrical signal. 40
- 45
28. The cultivation condition detecting device as claimed in claim 27, wherein said gear systems includes a first gear which is fixed to said support pin, and a second gear which has a smaller number of teeth than said first gear and is fixed to said rotary sensor, said two gears being engaged with each other to thereby transmit the rotational motion of said first gear mechanism to said rotary sensor while amplifying the rotational motion. 50
- 55
29. The cultivation condition detecting device as claimed in claim 27, wherein said cultivating machine comprises a plow machine, and the varia-

FIG. 1

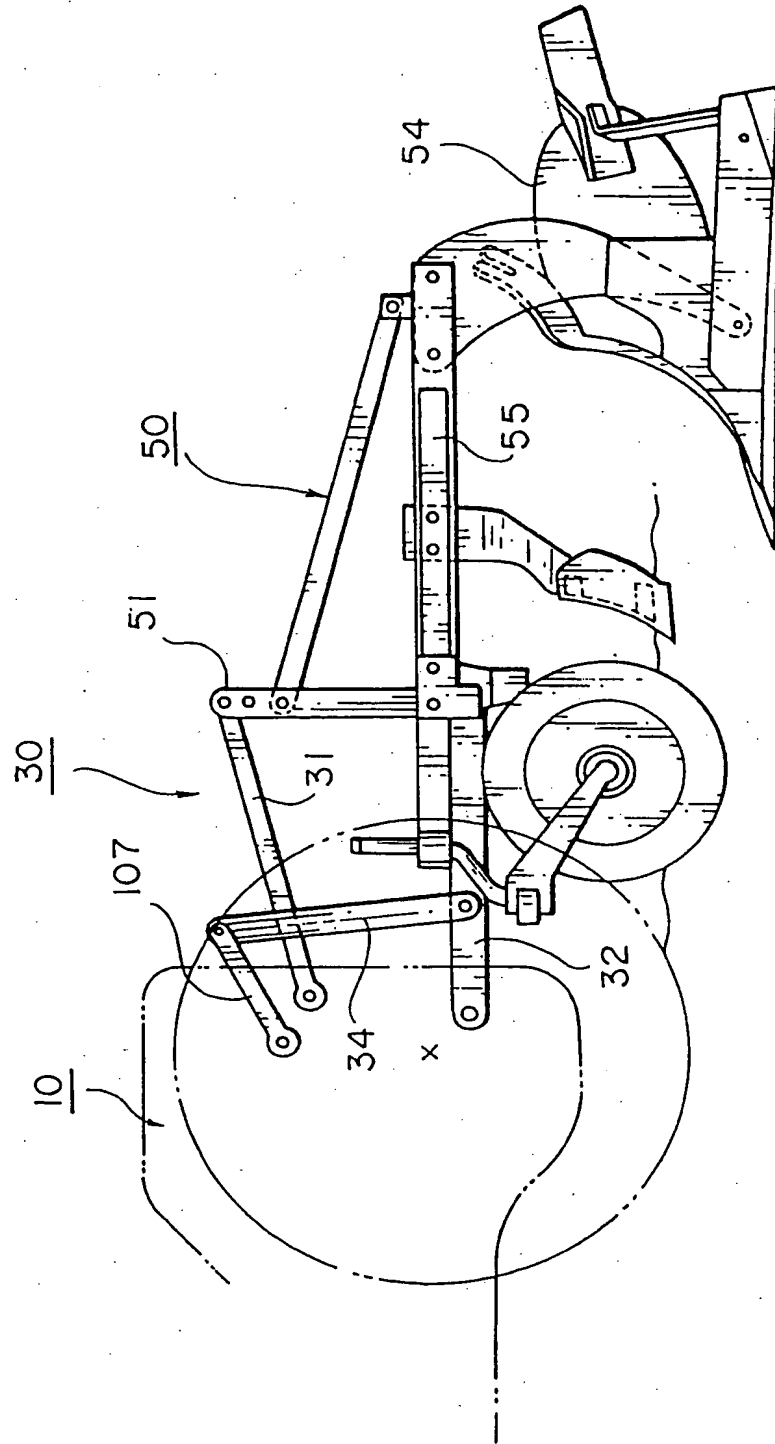




FIG. 4

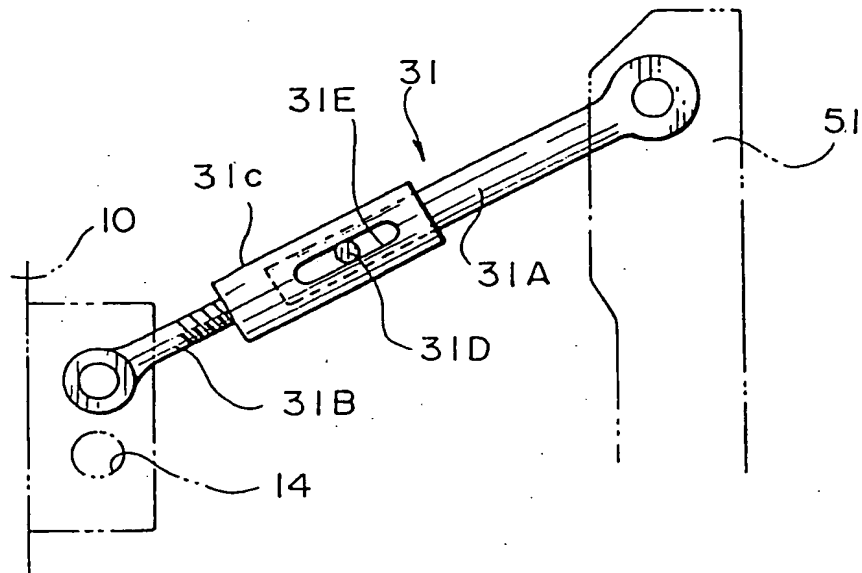


FIG. 5

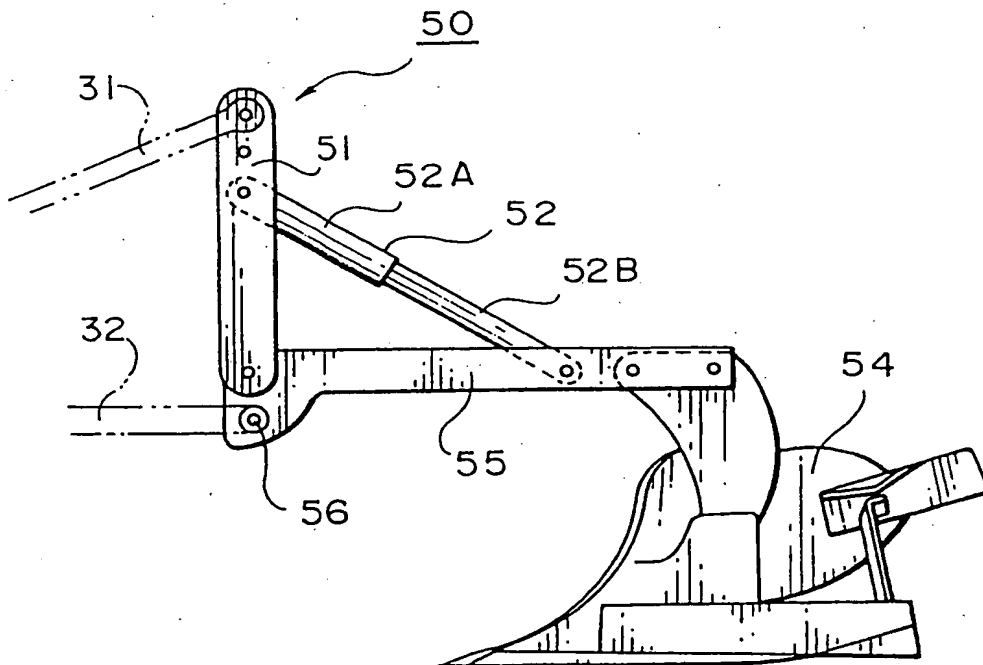




FIG. 6

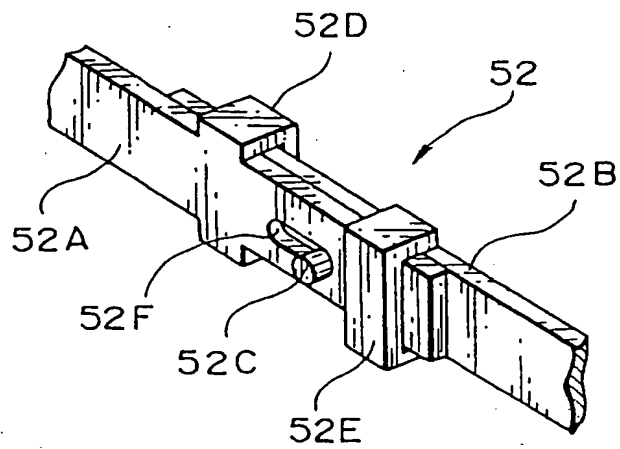
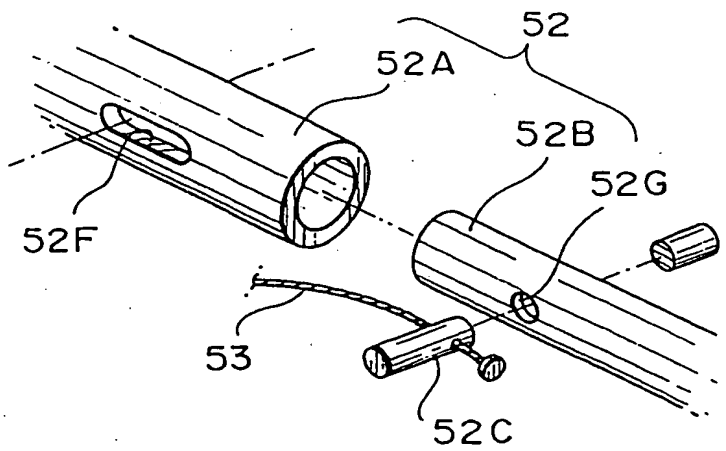
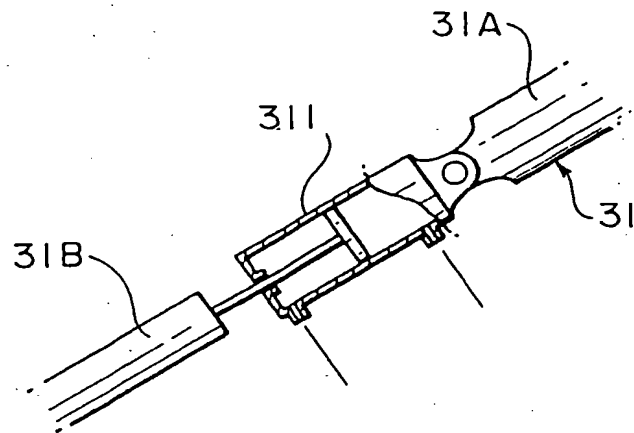


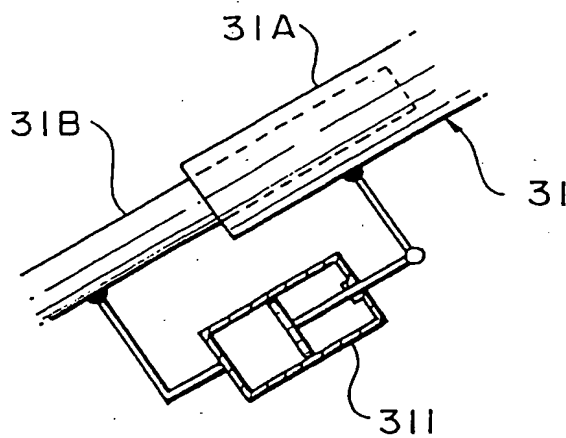
FIG. 7



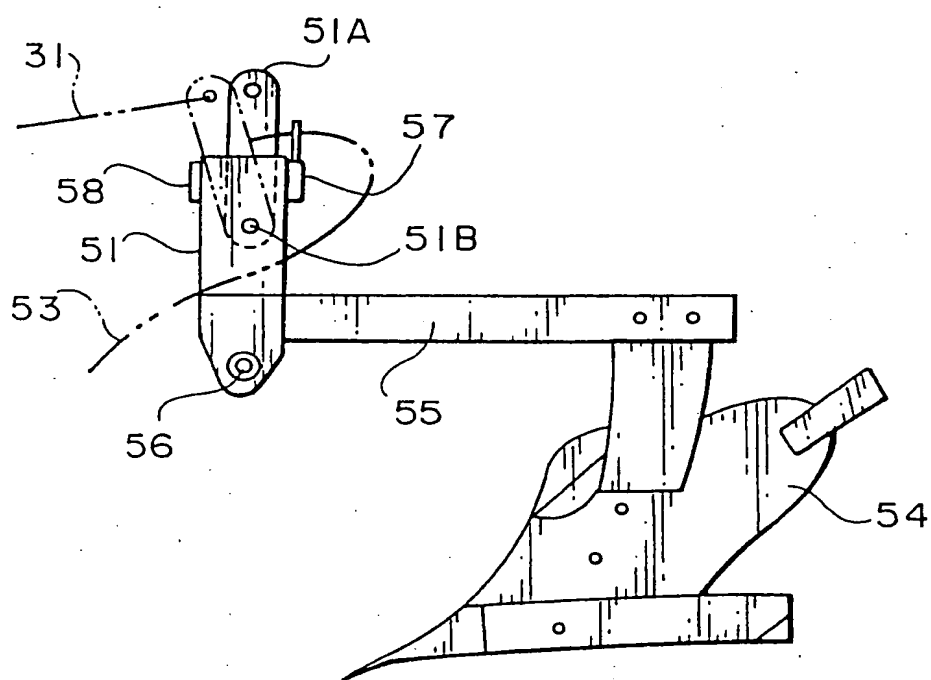
F I G . 8



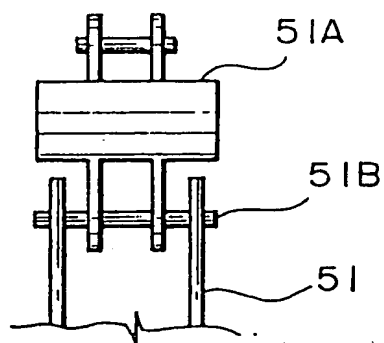
F I G . 9



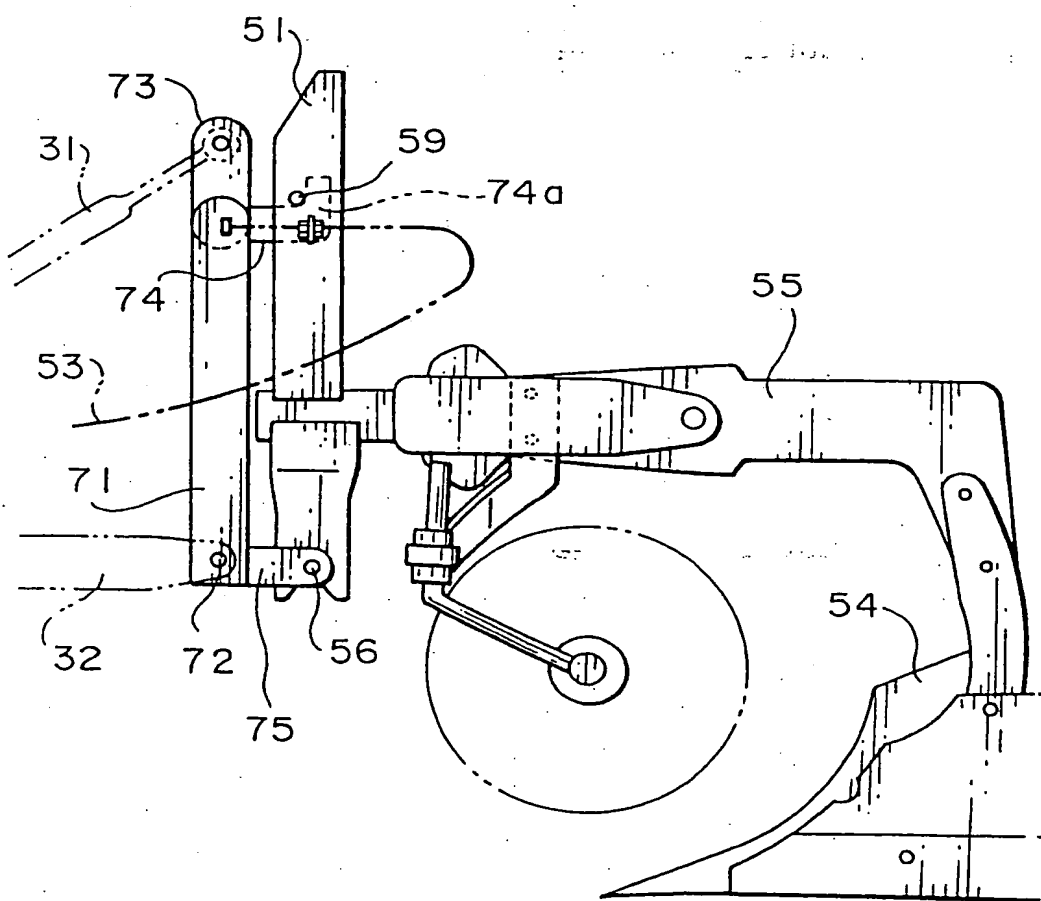
F I G . 10



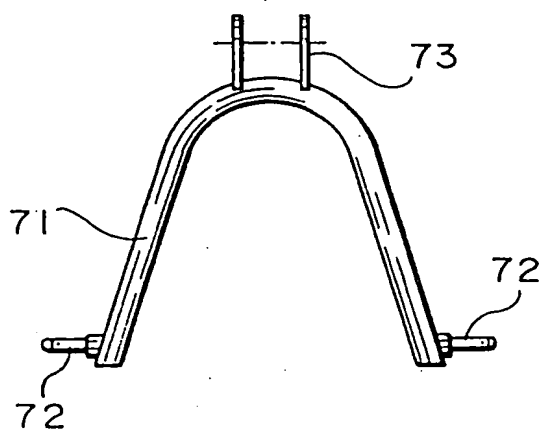
F I G . 11



F I G . 12



F I G . 13



F I G . 14

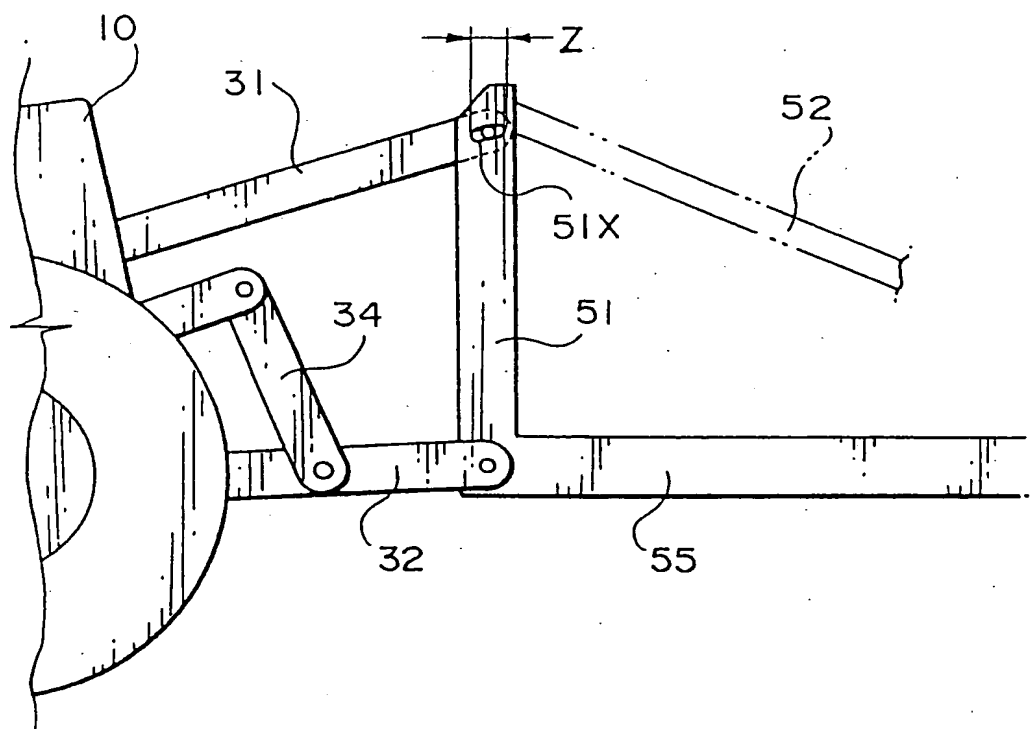
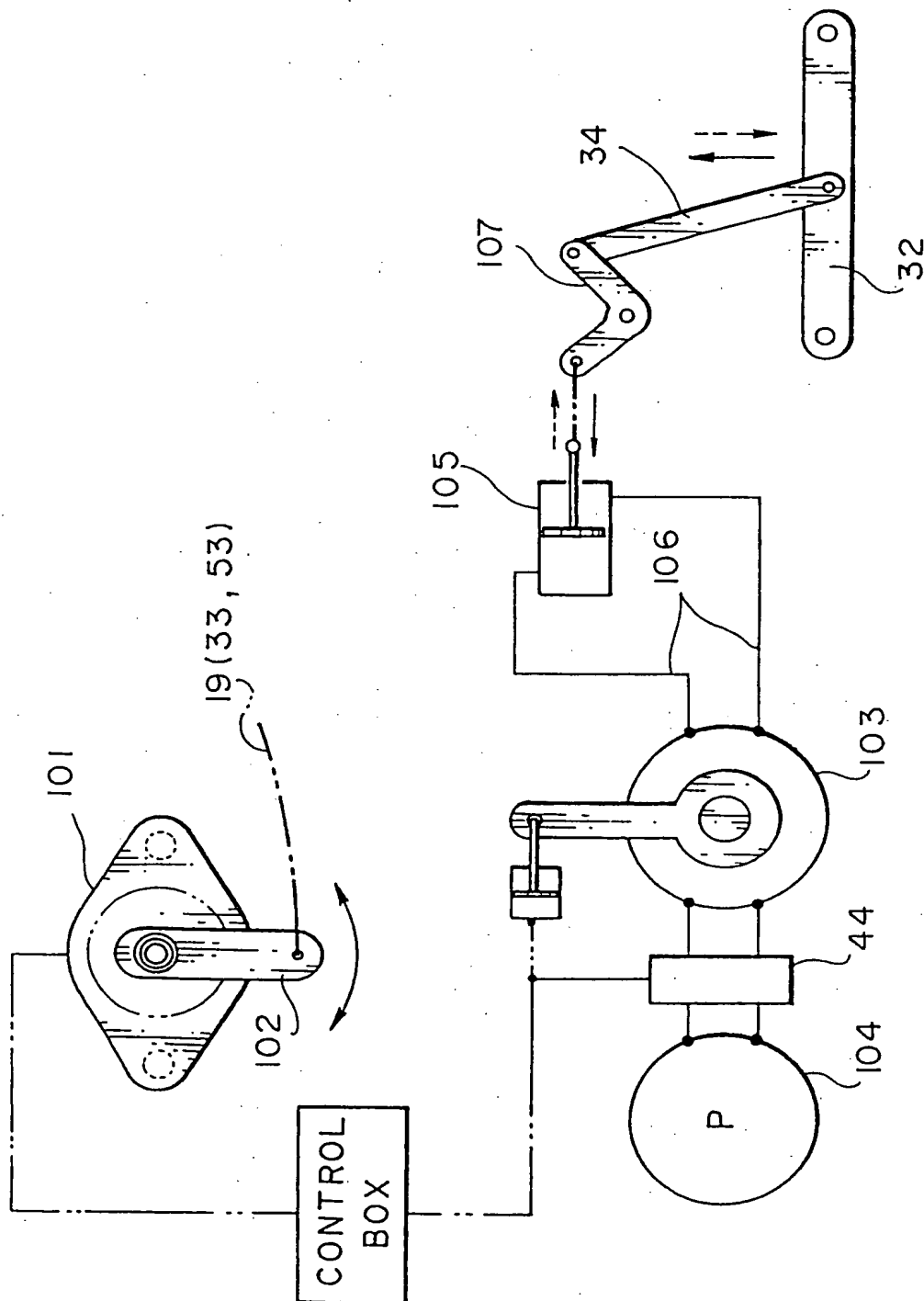
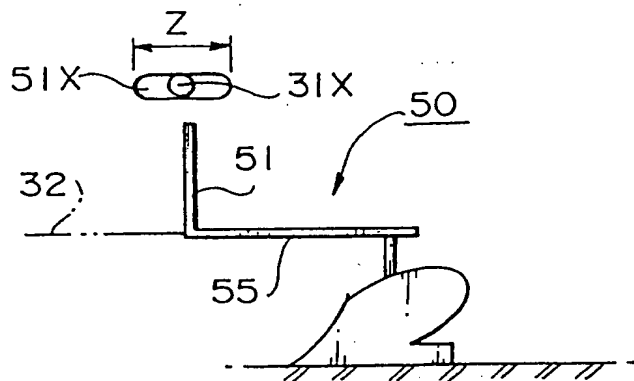


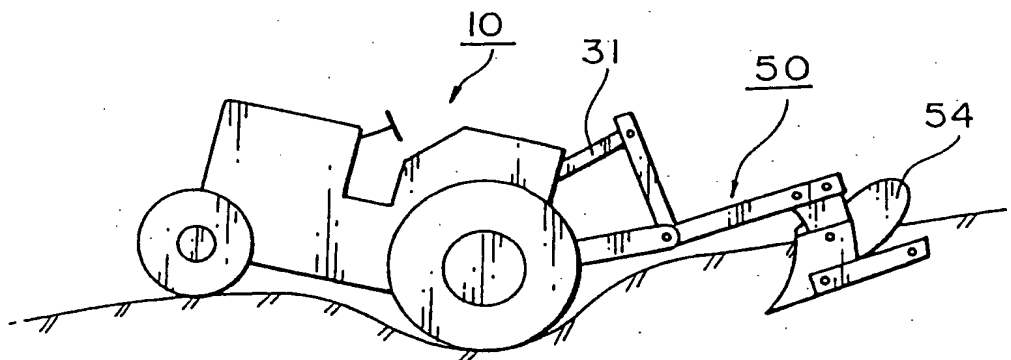
FIG. 15



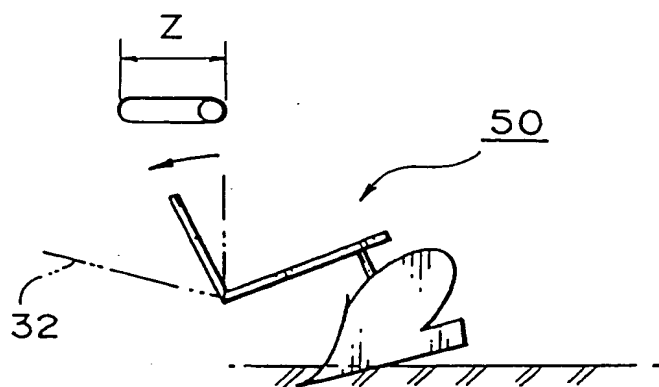
F I G . 16



F I G . 17A



F I G . 17 B



F I G . 18 A

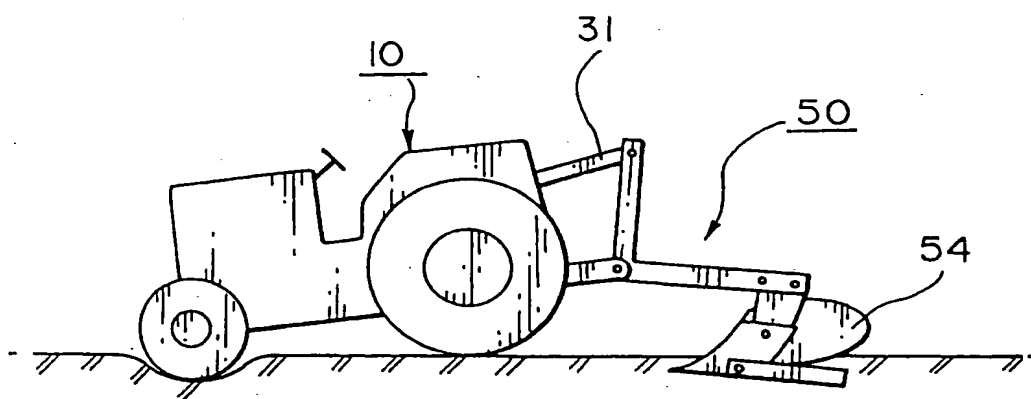




FIG. 18B

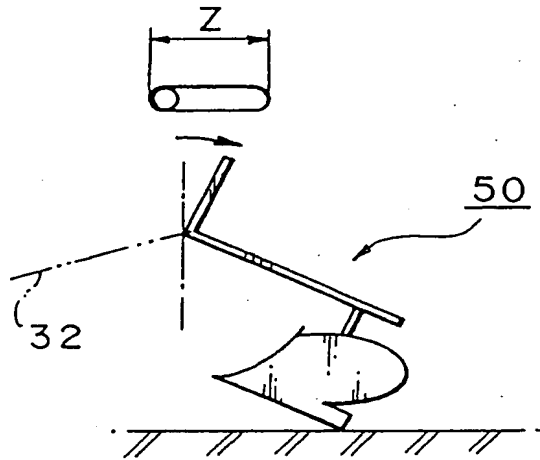
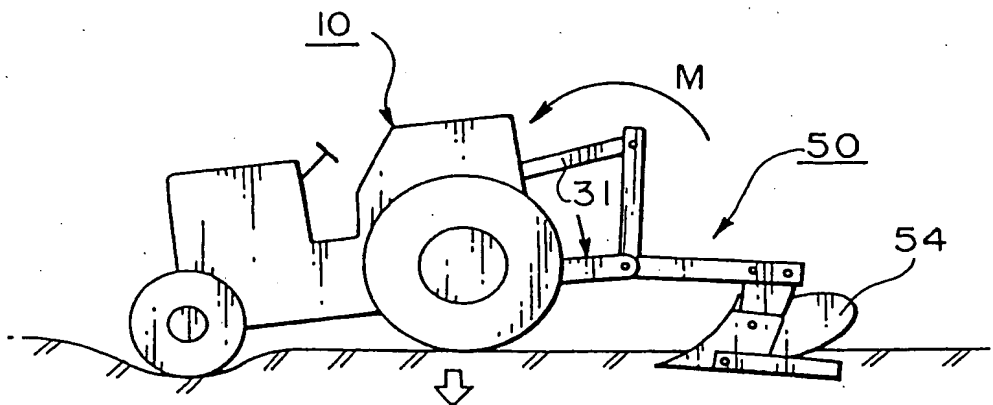


FIG. 19



F I G . 20

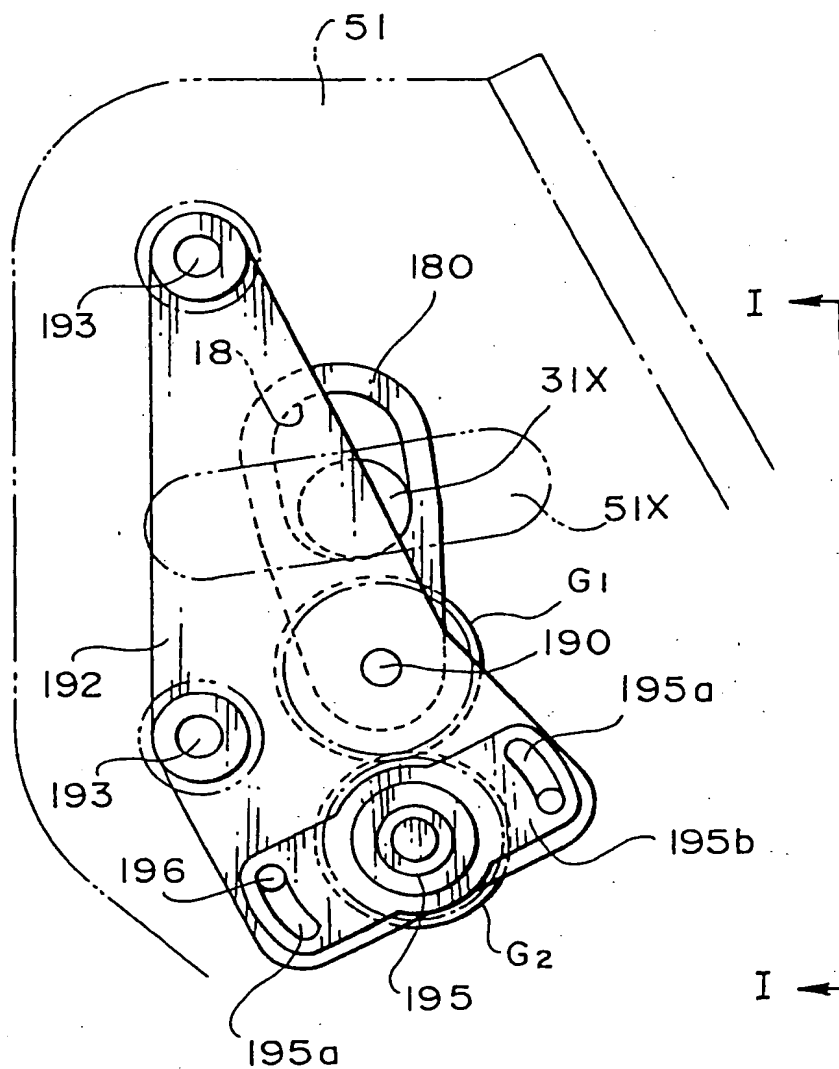


FIG. 21

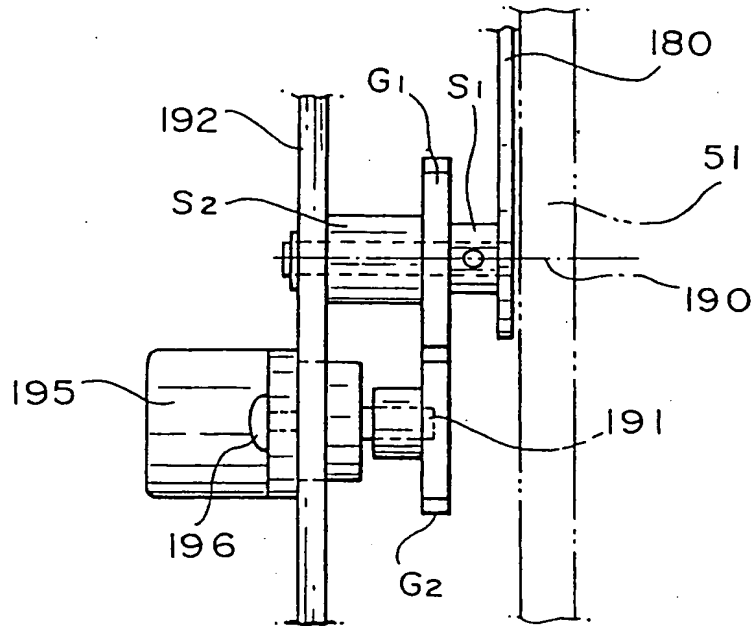


FIG. 23

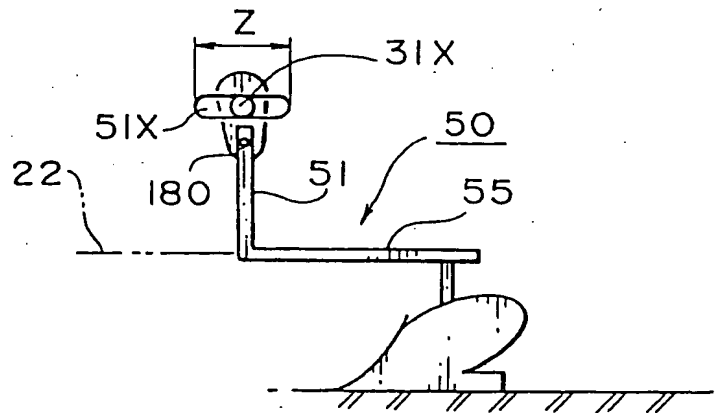
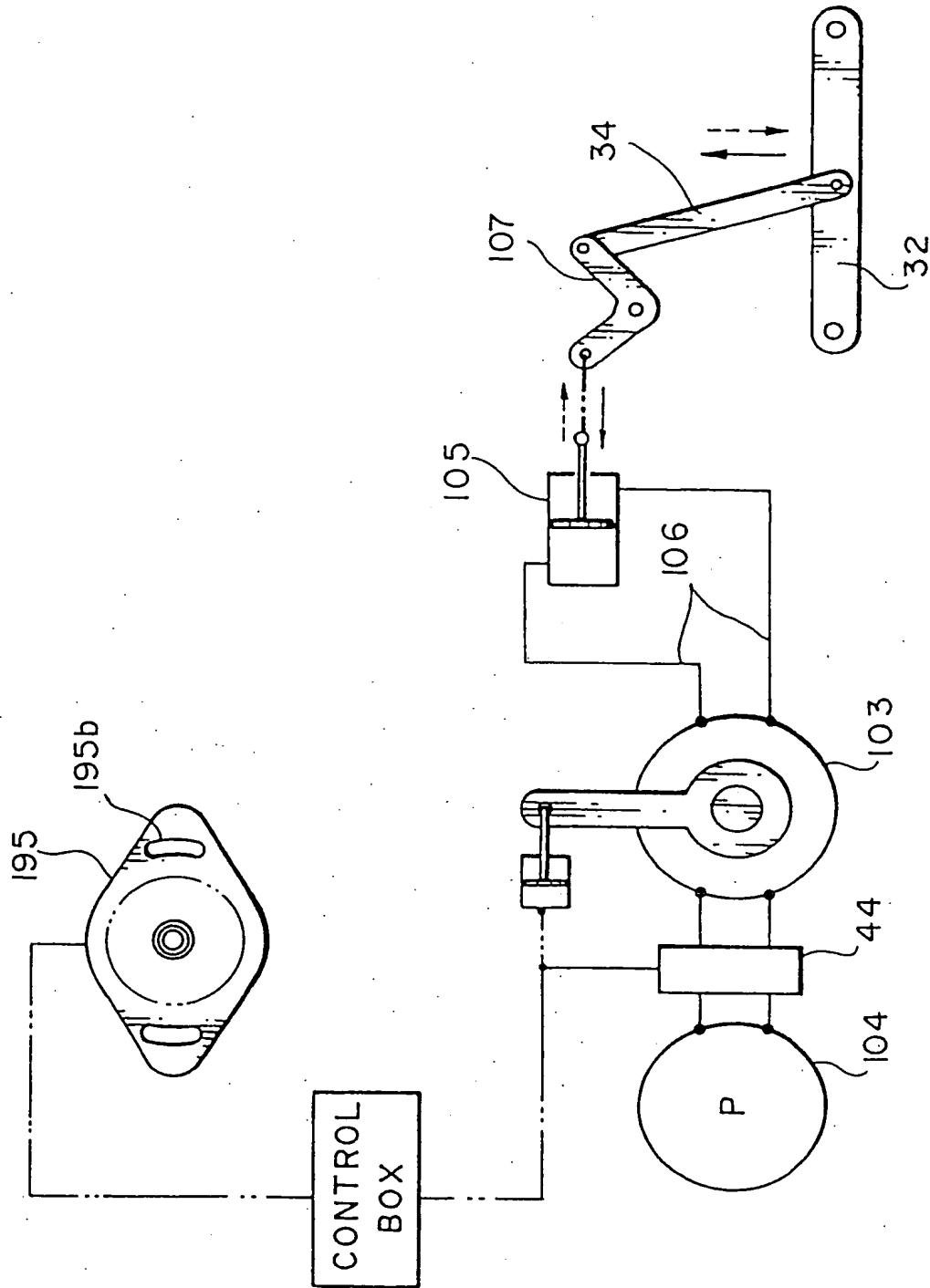
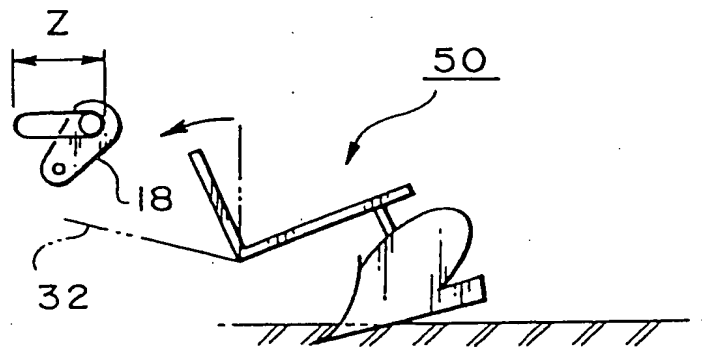


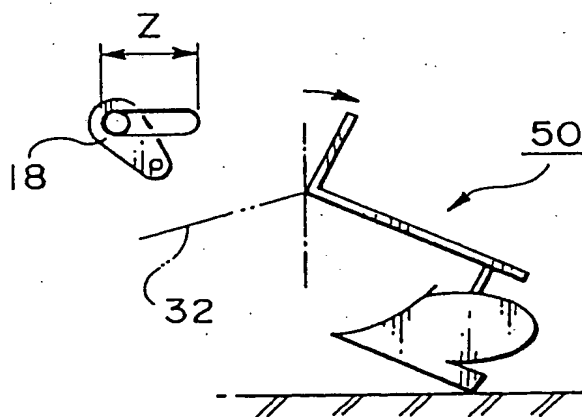
FIG. 22



F I G . 24



F I G . 25





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## EUROPEAN SEARCH REPORT

Application Number  
EP 94 30 7715

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.6)
X	GB-A-1 094 226 (FENDT)  * page 1, line 13 - line 68; figures * ---	1-3, 9, 12-14, 17-20, 23, 26	A01B63/111
X	GB-A-801 625 (INTERNATIONAL HARVESTER) * figure *	20-26	
A	---	5	
X	US-A-3 106 253 (SILVER) * figures *	20-26	
X	US-A-2 729 156 (WILLEY) * figures *	20-26	
A	---	4, 8	
X	FR-A-1 356 188 (KVERNELANDS) * page 2, left column, line 16 - right column, line 40; figures *	20, 23	
Y	---	1	TECHNICAL FIELDS SEARCHED (Int.Cl.6)
X	US-A-2 663 239 (RAPP) * figures *	20-26	A01B
Y	---	1	
X	US-A-2 722 874 (BOPF) * column 8, line 3 - line 8; figure 1 *	1	
A	FR-A-1 577 450 (KHD) * figure *	13	
A	DE-A-40 01 495 (WILDNER) * figure *	1	
A	DE-B-11 64 729 (INTERNATIONAL HARVESTER) * figure *	1	
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 6 April 1995	Examiner WALVOORT, B
<p><b>CATEGORY OF CITED DOCUMENTS</b></p> <p>X : particularly relevant if taken alone  Y : particularly relevant if combined with another document of the same category  A : technological background  O : non-written disclosure  P : intermediate document</p> <p>T : theory or principle underlying the invention  E : earlier patent document, but published on, or after the filing date  D : document cited in the application  I : document cited for other reasons  &amp; : member of the same patent family, corresponding document</p>			

EPO FORM 1503 03.92 (PXC01)



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## CLAIMS INCURRING FEES

The present European patent application comprised at the time of filing more than ten claims.

- ☐ All claims fees have been paid within the prescribed time limit. The present European search report has been drawn up for all claims.
- ☐ Only part of the claims fees have been paid within the prescribed time limit. The present European search report has been drawn up for the first ten claims and for those claims for which claims fees have been paid, namely claims:
- ☐ No claims fees have been paid within the prescribed time limit. The present European search report has been drawn up for the first ten claims.

## LACK OF UNITY OF INVENTION

The Search Division considers that the present European patent application does not comply with the requirement of unity of invention and relates to several inventions or groups of inventions, namely:

1. Claims 1-14, 17-26 and 15, 16 as far as technically dependent on claim 1: Cultivating system with free-motion mechanism
2. Claims 27-29: Cultivating condition detecting device with gear actuated sensor

- ☐ All further search fees have been paid within the fixed time limit. The present European search report has been drawn up for all claims.
- ☐ Only part of the further search fees have been paid within the fixed time limit. The present European search report has been drawn up for those parts of the European patent application which relate to the inventions in respects of which search fees have been paid, namely claims:
- ☒ None of the further search fees has been paid within the fixed time limit. The present European search report has been drawn up for those parts of the European patent application which relate to the invention first mentioned in the claims,

namely claims: 1-14, 17-26 and 15, 16 as far as technically dependent on claim 1